



# Structural-Parametric Models and Transfer Functions of Electromagnetoelastic Actuators Nano- and Microdisplacement for Mechatronic Systems

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**Abstract:** Structural-parametric models, parametric structural schematic diagrams and transfer functions of electromagnetoelastic actuators are determined. A generalized parametric structural schematic diagram of the electromagnetoelastic actuator is constructed. Effects of geometric and physical parameters of actuators and external load on its dynamic characteristics are determined. For calculations the mechatronic systems with piezoactuators for nano- and microdisplacement the parametric structural schematic diagrams and the transfer functions of piezoactuators are obtained.

**Keywords:** Electromagnetoelastic Actuator, Deformation, Nano- and Microdisplacement, Structural-Parametric Model, Piezoactuator, Parametric Structural Schematic Diagram, Transfer Function

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## 1. Introduction

Precise electromechanical actuators for mechatronic systems operate on electromagnetoelasticity (piezoelectric, piezomagnetic, electrostriction, and magnetostriction effects). Its application is promising in nanotechnology, nanobiology, power engineering, microelectronics, astronomy for large compound telescopes, antennas satellite telescopes and adaptive optics equipment for precision matching, compensation of temperature and gravitation deformations, and atmospheric turbulence via wave front correction. Piezoactuator - piezomechanical device intended for actuation of mechanisms, systems or management based on the piezoelectric effect, converts electrical signals into mechanical movement and force [1–5].

In the present paper is solving the problem of building the structural parametric model of the electromagnetoelastic actuator in contrast its electrical equivalent circuit [6, 7].

The piezoactuator of nanometric movements operates based on the inverse piezoeffect, in which the motion is achieved due to deformation of the piezoelement when an external electric voltage is applied to it. Piezoactuators for drives of nano- and micrometric movements provide a movement range from several nanometers to tens of microns, a sensitivity of up to 10

nm/V, a loading capacity of up to 1000 N, the power at the output shaft of up to 100 W, and a transmission band of up to 1000 Hz. The investigation of static and dynamic characteristics of a piezoactuator is necessary for calculation mechatronic systems of nano- and micrometric movements. At the nano- and microlevels, piezoactuators are used in linear nano- and microdrives and micropumps [8–11].

Piezoactuators provide high stress and speed of operation and return to the initial state when switched off; they have very low displacements. Piezoactuators are used in the majority of nanomanipulators for scanning tunneling microscopes (STMs), scanning force microscopes (SFM), and atomic force microscopes (AFMs). Nanorobotic manipulators with nano- and microdisplacements with piezoactuators based are a key component in nano- and microdisplacement nanorobotic systems. The main requirement for nanomanipulators is to guarantee the positioning accurate to nanometers [12–14].

By solving the wave equation with allowance methods of mathematical physics for the corresponding equation of electromagnetoelasticity, the boundary conditions on loaded working surfaces of a electromagnetoelastic actuator, and the strains along the coordinate axes, it is possible to construct a structural parametric model of the actuator. The transfer functions and the parametric structural schematic diagram of

the electromagnetoelastic actuator are obtained from a set of equations describing the corresponding structural parametric model of the electromagnetoelastic actuator for the mechatronic system [15–27].

## 2. Structural-Parametric Models and Parametric Structural Schematic Diagrams of Electromagnetoelastic Actuator

In the piezoactuator there are six stress components  $T_1, T_2, T_3, T_4, T_5, T_6$ , the components  $T_1 - T_3$  are related to extension-compression stresses,  $T_4 - T_6$  to shear stresses.

The matrix state equations [7] connecting the electric and elastic variables for polarized ceramics have the form

$$\mathbf{D} = \mathbf{d}\mathbf{T} + \boldsymbol{\varepsilon}^T \mathbf{E}, \quad (1)$$

$$\mathbf{S} = \mathbf{s}^E \mathbf{T} + \mathbf{d}' \mathbf{E}. \quad (2)$$

Here, the first equation describes the direct piezoelectric effect, and the second - the inverse piezoelectric effect;  $\mathbf{S}$  is the column matrix of relative deformations;  $\mathbf{T}$  is the column matrix of mechanical stresses;  $\mathbf{E}$  is the column matrix of electric field strength along the coordinate axes;  $\mathbf{D}$  is the column matrix of electric induction along the coordinate axes;  $\mathbf{s}^E$  is the elastic compliance matrix for  $E = \text{const}$ ; and  $\mathbf{d}'$  is the the transposed matrix of the piezoelectric modules.

In polarized ceramics PZT there are five independent components  $s_{11}^E, s_{12}^E, s_{13}^E, s_{33}^E, s_{55}^E$  in the elastic compliance matrix, three independent components  $d_{33}, d_{31}, d_{15}$  in the transposed matrix of the piezoelectric modules and three independent components  $\varepsilon_{11}^T, \varepsilon_{22}^T, \varepsilon_{33}^T$  in the matrix of dielectric constants.

The equation of electromagnetoelasticity of the actuator [7] has the form

$$S_i = s_{ij}^{E,H,\Theta} T_j + d_{mi}^{H,\Theta} E_m + d_{mi}^{E,\Theta} H_m + \alpha_i^{E,H} \Delta\Theta, \quad (3)$$

where  $S_i$  is the relative deformation along the axis  $i$ ,  $E$  is the electric field strength,  $H$  is the magnetic field strength,  $\Theta$  is the temperature,  $s_{ij}^{E,H,\Theta}$  is the elastic compliance for  $E = \text{const}$ ,  $H = \text{const}$ ,  $\Theta = \text{const}$ ,  $T_j$  is the mechanical stress along the axis  $j$ ,  $d_{mi}^{H,\Theta}$  is the piezomodule, i.e., the partial derivative of the relative deformation with respect to the electric field strength for constant magnetic field strength and temperature, i.e., for  $H = \text{const}$ ,  $\Theta = \text{const}$ ,  $E_m$  is the electric field strength along the axis  $m$ ,  $d_{mi}^{E,\Theta}$  is the magnetostriction coefficient,  $H_m$  is the magnetic field strength along the axis  $m$ ,  $\alpha_i^{E,H}$  is the coefficient of thermal expansion,  $\Delta\Theta$  is deviation of the temperature  $\Theta$

from the value  $\Theta = \text{const}$ ,  $i = 1, 2, \dots, 6$ ,  $j = 1, 2, \dots, 6$ ,  $m = 1, 2, 3$ .

When the electric and magnetic fields act on the electromagnetoelastic actuator separately, we have the respective electromagnetoelasticity equations [7] as the equations of inverse piezoelectric effect:

$S_3 = d_{33} E_3 + s_{33}^E T_3$  for the longitudinal deformation when the electric field along axis 3 causes deformation along axis 3,

$S_1 = d_{31} E_3 + s_{11}^E T_1$  for the transverse deformation when the electric field along axis 3 causes deformation along axis 1,

$S_5 = d_{15} E_1 + s_{55}^E T_5$  for the shift deformation when the electric field along axis 1 causes deformation in the plane perpendicular to this axis,

as the equations of magnetostriction:

$S_3 = d_{33} H_3 + s_{33}^H T_3$  for the longitudinal deformation when the magnetic field along axis 3 causes deformation along axis 3,

$S_1 = d_{31} H_3 + s_{11}^H T_1$  for the transverse deformation when the magnetic field along axis 3 causes deformation along axis 1,

$S_5 = d_{15} H_1 + s_{55}^H T_5$  for the shift deformation when the magnetic field along axis 1 causes deformation in the plane perpendicular to this axis.

To illustrate this, we consider piezoelectric problems. Let us consider the longitudinal piezoelectric effect in a piezoelectric actuator shown in Fig. 1, which represents a piezoelectric plate of thickness  $\delta$  with the electrodes deposited on its faces perpendicular to axis 3, the area of which is equal to  $S_0$ . The direction of the polarization axis  $P$ , i.e., the direction along which polarization was performed, is usually taken as the direction of axis 3.

The equation of the inverse piezoelectric effect [5, 7] for the longitudinal strain in a voltage-controlled piezoactuator has the following form:

$$S_3 = d_{33} E_3(t) + s_{33}^E T_3(x, t), \quad (4)$$

Here,  $S_3 = \partial \xi(x, t) / \partial x$  is the relative displacement of the cross section of the piezoactuator,  $d_{33}$  is the piezoelectric module for the longitudinal piezoelectric effect,  $E_3(t) = U(t) / \delta$  is the electric field strength,  $U(t)$  is the voltage between the electrodes of actuator,  $\delta$  is the thickness,  $s_{33}^E$  is the elastic compliance along axis 3, and  $T_3$  is the mechanical stress along axis 3.

The equation of equilibrium for the forces acting on the piezoactuator (piezoelectric plate) Figure 1 can be written as

$$T_3 S_0 = F + M \frac{\partial^2 \xi(x, t)}{\partial t^2} \quad (5)$$

where  $F$  is the external force applied to the piezoactuator,  $S_0$  is the cross section area and  $M$  is the displaced mass.

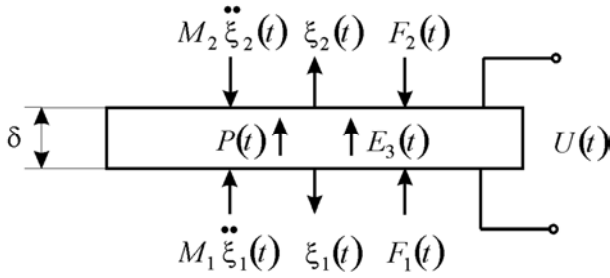


Fig. 1. Piezoactuator for the longitudinal piezoelectric effect.

For constructing a structural parametric model of the voltage-controlled piezoactuator, let us solve simultaneously the wave equation, the equation of the inverse longitudinal piezoelectric effect, and the equation of forces acting on the faces of the piezoactuator. Calculations of the piezoactuators are performed using a wave equation [3–7, 24] describing the wave propagation in a long line with damping but without distortions, which can be written as

$$\frac{1}{(c^E)^2} \frac{\partial^2 \xi(x, t)}{\partial t^2} + \frac{2\alpha}{c^E} \frac{\partial \xi(x, t)}{\partial t} + \alpha^2 \xi(x, t) = \frac{\partial^2 \xi(x, t)}{\partial x^2}, \quad (6)$$

where  $\xi(x, t)$  is the displacement of the section of the piezoactuator,  $x$  is the coordinate,  $t$  is time,  $c^E$  is the sound speed for  $E = \text{const}$ ,  $\alpha$  is the damping coefficient. Using the Laplace transform, we can reduce the original problem for the partial differential hyperbolic equation of type (6) to a simpler problem for the linear ordinary differential equation [8, 9] with the parameter of the Laplace operator  $p$ .

Applying the Laplace transform to the wave equation (6)

$$\Xi(x, p) = L\{\xi(x, t)\} = \int_0^\infty \xi(x, t) e^{-pt} dt, \quad (7)$$

and setting the zero initial conditions,

$$\xi(x, t)|_{t=0} = \frac{\partial \xi(x, t)}{\partial t}|_{t=0} = 0. \quad (8)$$

We obtain the linear ordinary second-order differential equation with the parameter  $p$  written as

$$\frac{d^2 \Xi(x, p)}{dx^2} - \left[ \frac{1}{(c^E)^2} p^2 + \frac{2\alpha}{c^E} p + \alpha^2 \right] \Xi(x, p) = 0, \quad (9)$$

with its solution being the function

$$\Xi(x, p) = Ce^{-x\gamma} + Be^{x\gamma}, \quad (10)$$

where  $\Xi(x, p)$  is the Laplace transform of the displacement of the section of the piezoelectric actuator,  $\gamma = p/c^E + \alpha$  is the propagation coefficient. Determining coefficients  $C$  and  $B$  from the boundary conditions as

$$\Xi(0, p) = \Xi_1(p) \text{ for } x = 0, \quad (11)$$

$$\Xi(\delta, p) = \Xi_2(p) \text{ for } x = \delta.$$

Then, the constant coefficients

$$C = (\Xi_1 e^{\delta\gamma} - \Xi_2) / [2\text{sh}(\delta\gamma)], \quad B = -(\Xi_1 e^{-\delta\gamma} - \Xi_2) / [2\text{sh}(\delta\gamma)]. \quad (12)$$

The solution (9) can be written as

$$\Xi(x, p) = \{\Xi_1(p) \text{sh}[(\delta - x)\gamma] + \Xi_2(p) \text{sh}(x\gamma)\} / \text{sh}(\delta\gamma). \quad (13)$$

The equations for the forces on the faces of the piezoactuator

$$T_3(0, p) S_0 = F_1(p) + M_1 p^2 \Xi_1(p) \text{ for } x = 0, \quad (14)$$

$$T_3(\delta, p) S_0 = -F_2(p) - M_2 p^2 \Xi_2(p) \text{ for } x = \delta,$$

where  $T_3(0, p)$  and  $T_3(\delta, p)$  are determined from the equation of the inverse piezoelectric effect.

For  $x = 0$  and  $x = \delta$ , we obtain the following set of equations for determining stresses in the piezoactuator:

$$T_3(0, p) = \frac{1}{s_{33}^E} \frac{d\Xi(x, p)}{dx} \Big|_{x=0} - \frac{d_{33}}{s_{33}^E} E_3(p), \quad (15)$$

$$T_3(\delta, p) = \frac{1}{s_{33}^E} \frac{d\Xi(x, p)}{dx} \Big|_{x=\delta} - \frac{d_{33}}{s_{33}^E} E_3(p).$$

Equations (14) yield the following set of equations for the structural parametric model of the piezoactuator Figure 2:

$$\begin{aligned} \Xi_1(p) &= [1 / (M_1 p^2)] \cdot \\ &\left\{ -F_1(p) + (1 / \chi_{33}^E) [d_{33} E_3(p) - [\gamma / \text{sh}(\delta\gamma)] [\text{ch}(\delta\gamma) \Xi_1(p) - \Xi_2(p)]] \right\}, \\ \Xi_2(p) &= [1 / (M_2 p^2)] \cdot \\ &\left\{ -F_2(p) + (1 / \chi_{33}^E) [d_{33} E_3(p) - [\gamma / \text{sh}(\delta\gamma)] [\text{ch}(\delta\gamma) \Xi_2(p) - \Xi_1(p)]] \right\}, \end{aligned} \quad (16)$$

where  $\chi_{33}^E = s_{33}^E / S_0$ .

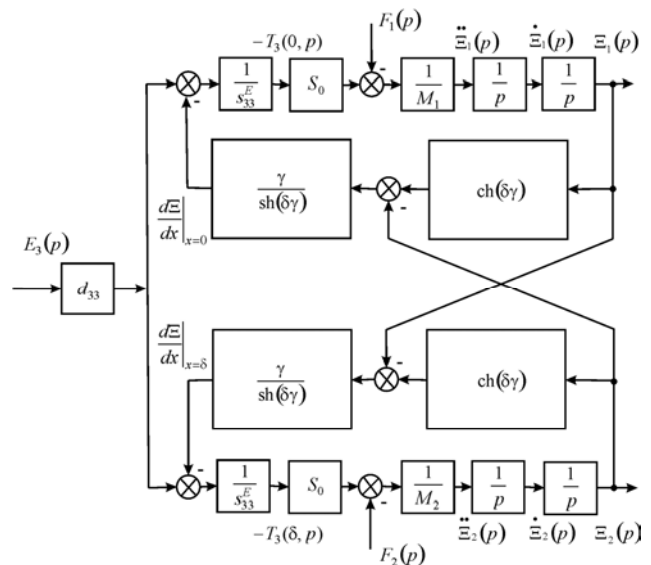


Fig. 2. Parametric structural schematic diagram of a voltage-controlled piezoactuator for longitudinal piezoelectric effect.

The equation of the inverse transverse piezoeffect [5, 7]

$$S_1 = d_{31}E_3(t) + s_{11}^E T_1(x, t), \quad (17)$$

where  $S_1 = \partial \xi(x, t) / \partial x$  is the relative displacement of the cross section along axis 1 Figure 3,  $d_{31}$  is the piezoelectric module for the transverse piezoeffect,  $s_{11}^E$  is the elastic compliance along axis 1,  $T_1$  is the stress along axis 1.

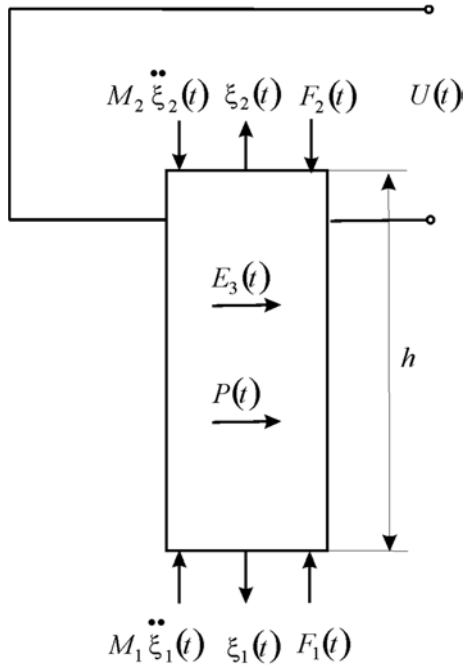


Fig. 3. Piezoactuator for the transverse piezoelectric effect.

The solution of the linear ordinary differential equation (9) can be written as (10), where the constants  $C$  and  $B$  in the form

$$\Xi(0, p) = \Xi_1(p) \text{ for } x = 0, \quad (18)$$

$$\Xi(l, p) = \Xi_2(p) \text{ for } x = h,$$

$$C = (\Xi_1 e^{h\gamma} - \Xi_2) / [2 \operatorname{sh}(h\gamma)], \quad B = -(\Xi_1 e^{-h\gamma} - \Xi_2) / [2 \operatorname{sh}(h\gamma)]. \quad (19)$$

Then, the solution (9) can be written as

$$\Xi(x, p) = \{\Xi_1(p) \operatorname{sh}[(h-x)\gamma] + \Xi_2(p) \operatorname{sh}(x\gamma)\} / \operatorname{sh}(h\gamma). \quad (20)$$

The equations of forces acting on the faces of the piezoactuator

$$T_1(0, p)S_0 = F_1(p) + M_1 p^2 \Xi_1(p) \text{ for } x = 0, \quad (21)$$

$$T_1(h, p)S_0 = -F_2(p) - M_2 p^2 \Xi_2(p) \text{ for } x = h,$$

where  $T_1(0, p)$  and  $T_1(h, p)$  are determined from the equation of the inverse piezoeffect. Thus, we obtain

$$T_1(0, p) = \frac{1}{s_{11}^E} \frac{d\Xi(x, p)}{dx} \Big|_{x=0} - \frac{d_{31}}{s_{11}^E} E_3(p), \quad (22)$$

$$T_1(h, p) = \frac{1}{s_{11}^E} \frac{d\Xi(x, p)}{dx} \Big|_{x=h} - \frac{d_{31}}{s_{11}^E} E_3(p).$$

The set of equations (16) for mechanical stresses in piezoactuator yields the following set of equations describing the structural parametric model and parametric structural schematic diagram of piezoactuator Figure 4

$$\Xi_1(p) = [1 / (M_1 p^2)] \cdot \{-F_1(p) + (1 / \chi_{11}^E) [d_{31} E_3(p) - [\gamma / \operatorname{sh}(h\gamma)] [\operatorname{ch}(h\gamma) \Xi_1(p) - \Xi_1(p)]]\}, \quad (23)$$

$$\Xi_2(p) = [1 / (M_2 p^2)] \cdot \{-F_2(p) + (1 / \chi_{11}^E) [d_{31} E_3(p) - [\gamma / \operatorname{sh}(h\gamma)] [\operatorname{ch}(h\gamma) \Xi_2(p) - \Xi_1(p)]]\},$$

where  $\chi_{11}^E = s_{11}^E / S_0$ .

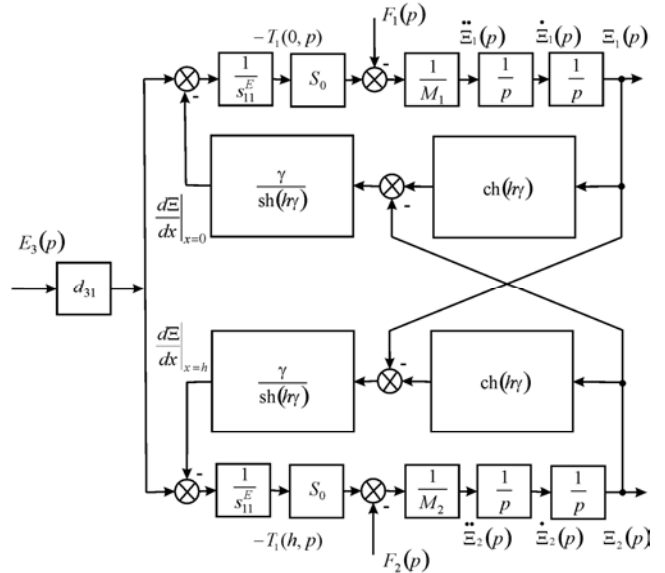


Fig. 4. Parametric structural schematic diagram of a voltage-controlled piezoactuator for transverse piezoelectric effect.

For piezoactuator of the shift piezoelectric effect Figure 5 we obtain the following set of equations describing the structural parametric model and parametric structural schematic diagram of piezoactuator Figure 6

$$\Xi_1(p) = [1 / (M_1 p^2)] \cdot \{-F_1(p) + (1 / \chi_{55}^E) [d_{15} E_1(p) - [\gamma / \operatorname{sh}(b\gamma)] [\operatorname{ch}(b\gamma) \Xi_1(p) - \Xi_1(p)]]\}, \quad (24)$$

$$\Xi_2(p) = [1 / (M_2 p^2)] \cdot \{-F_2(p) + (1 / \chi_{55}^E) [d_{15} E_1(p) - [\gamma / \operatorname{sh}(b\gamma)] [\operatorname{ch}(b\gamma) \Xi_2(p) - \Xi_1(p)]]\},$$

where  $\chi_{55}^E = s_{55}^E / S_0$ .

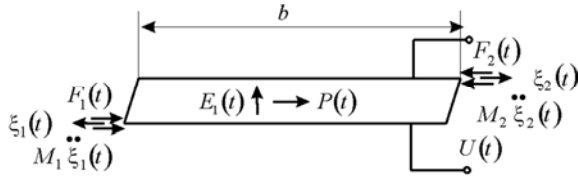


Fig. 5. Piezoactuator for the shift piezoelectric effect.

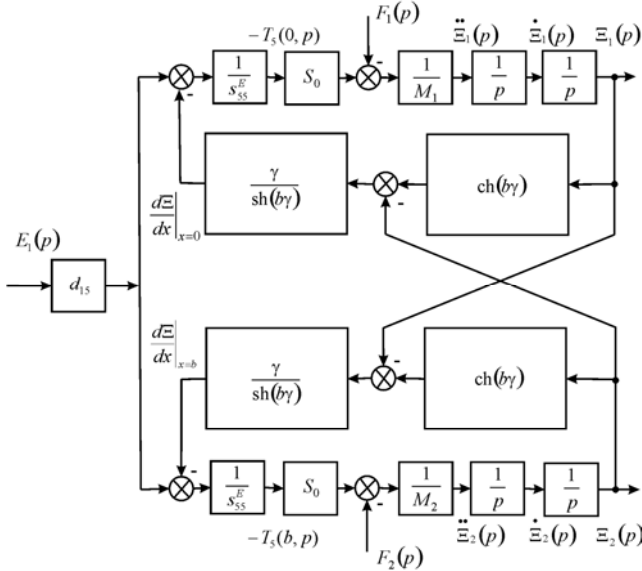


Fig. 6. Parametric structural schematic diagram of a voltage-controlled piezoactuator for shift piezoelectric effect.

We obtain the system of equations describing the generalized structural-parametric model of the electromagnetoelastic actuator taking into account (3) generalized electromagnetoelasticity equation in the following form

$$\Xi_1(p) = [1/(M_1 p^2)] \cdot \left\{ -F_1(p) + (1/\chi_{ij}^\Psi) [v_{mi} \Psi_m(p) - [\gamma/\text{sh}(l\gamma)] [\text{ch}(l\gamma)\Xi_1(p) - \Xi_2(p)]] \right\}, \quad (25)$$

$$\Xi_2(p) = [1/(M_2 p^2)] \cdot \left\{ -F_2(p) + (1/\chi_{ij}^\Psi) [v_{mi} \Psi_m(p) - [\gamma/\text{sh}(l\gamma)] [\text{ch}(l\gamma)\Xi_2(p) - \Xi_1(p)]] \right\},$$

$$\text{where } v_{mi} = \begin{Bmatrix} d_{33}, d_{31}, d_{15} \\ g_{33}, g_{31}, g_{15} \\ d_{33}, d_{31}, d_{15} \end{Bmatrix}, \quad \Psi_m = \begin{Bmatrix} E_3, E_1 \\ D_3, D_1 \\ H_3, H_1 \end{Bmatrix}, \quad s_{ij}^\Psi = \begin{Bmatrix} s_{33}^E, s_{11}^E, s_{55}^E \\ s_{33}^D, s_{11}^D, s_{55}^D \\ s_{33}^H, s_{11}^H, s_{55}^H \end{Bmatrix},$$

$$c^\Psi = \begin{Bmatrix} c^E \\ c^D \\ c^H \end{Bmatrix}, \quad \gamma = \begin{Bmatrix} \gamma^E \\ \gamma^D \\ \gamma^H \end{Bmatrix}, \quad l = \begin{Bmatrix} h \\ b \end{Bmatrix}, \quad \chi_{ij}^\Psi = s_{ij}^\Psi / S_0,$$

then parameters  $\Psi$  of the control for the electromagnetoelastic actuator:  $E$  for voltage control,  $D$  for current control,  $H$  for magnetic field strength control. Figure 7 shows the generalized parametric block diagram of the electromagnetoelastic actuator corresponding to the set of equations (25).

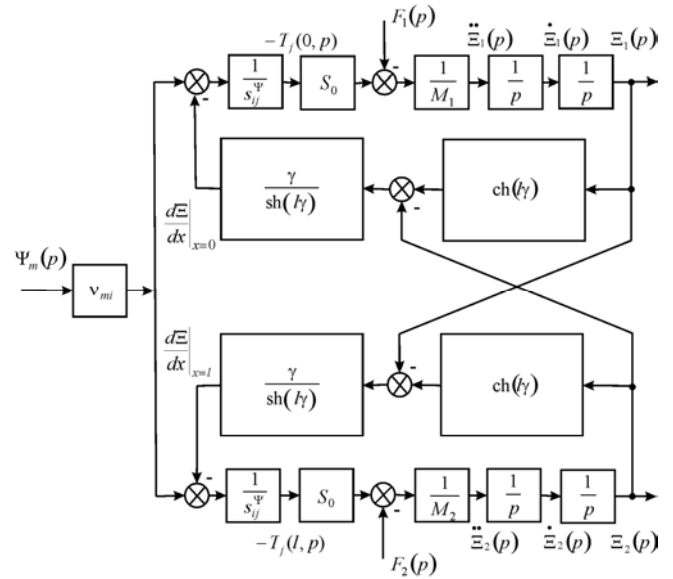


Fig. 7. Generalized parametric structural schematic diagram of the electromagnetoelastic actuator.

### 3. Transfer Functions of Electromagnetoelastic Actuator

Generalized structural-parametric model (25) of the actuator after algebraic transformations provides the transfer functions of the actuator in matrix form, where the transfer functions are the ratio of the Laplace transform of the displacement of the face and the Laplace transform of the corresponding parameter or force at zero initial conditions.

$$\Xi_1(p) = W_{11}(p)\Psi_m(p) + W_{12}(p)F_1(p) + W_{13}(p)F_2(p), \quad (26)$$

$$\Xi_2(p) = W_{21}(p)\Psi_m(p) + W_{22}(p)F_1(p) + W_{23}(p)F_2(p),$$

where the generalized transfer functions of the electromagnetoelastic actuator are

$$W_{11}(p) = \Xi_1(p)/\Psi_m(p) = v_{mi} [M_2 \chi_{ij}^\Psi p^2 + \gamma \text{th}(l\gamma/2)] / A_{ij},$$

$$\chi_{ij}^\Psi = s_{ij}^\Psi / S_0,$$

$$A_{ij} = M_1 M_2 (\chi_{ij}^\Psi)^2 p^4 + \{ (M_1 + M_2) \chi_{ij}^\Psi / [c^\Psi \text{th}(l\gamma)] \} p^3 + \{ (M_1 + M_2) \chi_{ij}^\Psi \alpha / \text{th}(l\gamma) + 1 / (c^\Psi)^2 \} p^2 + 2\alpha p / c^\Psi + \alpha^2,$$

$$W_{21}(p) = \Xi_2(p)/\Psi_m(p) = v_{mi} [M_1 \chi_{ij}^\Psi p^2 + \gamma \text{th}(l\gamma/2)] / A_{ij},$$

$$W_{12}(p) = \Xi_1(p)/F_1(p) = -\chi_{ij}^\Psi [M_2 \chi_{ij}^\Psi p^2 + \gamma / \text{th}(l\gamma)] / A_{ij},$$

$$W_{13}(p) = \Xi_1(p)/F_2(p) =$$

$$W_{22}(p) = \Xi_2(p)/F_1(p) = [\chi_{ij}^\Psi \gamma / \text{sh}(l\gamma)] / A_{ij},$$

$$W_{23}(p) = \Xi_2(p)/F_2(p) = -\chi_{ij}^\Psi [M_1 \chi_{ij}^\Psi p^2 + \gamma / \text{th}(l\gamma)] / A_{ij}.$$

Therefore, we obtain from equations (26) the generalized matrix equation for the electromagnetoelastic actuator in the matrix form for mechatronic systems

$$\begin{pmatrix} \Xi_1(p) \\ \Xi_2(p) \end{pmatrix} = \begin{pmatrix} W_{11}(p) & W_{12}(p) & W_{13}(p) \\ W_{21}(p) & W_{22}(p) & W_{23}(p) \end{pmatrix} \begin{pmatrix} \Psi_m(p) \\ F_1(p) \\ F_2(p) \end{pmatrix}. \quad (27)$$

Let us find the displacement of the faces the electromagnetoelastic actuator in a stationary regime for  $\Psi_m(t) = \Psi_{m0} \cdot 1(t)$ ,  $F_1(t) = F_2(t) = 0$  and inertial load. The static displacement of the faces the electromagnetoelastic actuator  $\xi_1(\infty)$  and  $\xi_2(\infty)$  can be written in the form:

$$\xi_1(\infty) = \lim_{t \rightarrow \infty} \xi_1(t) = \lim_{p \rightarrow 0} p W_{11}(p) \Psi_{m0} / p = \lim_{\alpha \rightarrow 0} \dots, \quad (28)$$

$$\nu_{mi} I \Psi_{m0} (M_2 + m/2) / (M_1 + M_2 + m) \quad (28)$$

$$\xi_2(\infty) = \lim_{t \rightarrow \infty} \xi_2(t) = \lim_{p \rightarrow 0} p W_{21}(p) \Psi_{m0} / p = \lim_{\alpha \rightarrow 0} \dots, \quad (29)$$

$$\nu_{mi} I \Psi_{m0} (M_1 + m/2) / (M_1 + M_2 + m) \quad (29)$$

$$\xi_1(\infty) + \xi_2(\infty) = \lim_{t \rightarrow \infty} (\xi_1(t) + \xi_2(t)) = \nu_{mi} I \Psi_{m0}, \quad (30)$$

where  $m$  is the mass of the electromagnetoelastic actuator,  $M_1, M_2$  are the load masses.

Let us consider a numerical example of the calculation of static characteristics of the piezoactuator from piezoceramics PZT under the longitudinal piezoelectric effect at  $m \ll M_1$  and  $m \ll M_2$ . For  $d_{33} = 4 \cdot 10^{-10}$  m/V,  $U = 250$  V,  $M_1 = 10$  kg and  $M_2 = 40$  kg we obtain the static displacement of the faces of the piezoactuator  $\xi_1(\infty) = 80$  nm,  $\xi_2(\infty) = 20$  nm,  $\xi_1(\infty) + \xi_2(\infty) = 100$  nm.

The static displacement the faces of the piezoactuator for the transverse piezoelectric effect and inertial load at  $U(t) = U_0 \cdot 1(t)$ ,  $E_3(t) = E_{30} \cdot 1(t) = (U_0/\delta) \cdot 1(t)$  and  $F_1(t) = F_2(t) = 0$  can be written in the following form:

$$\xi_1(\infty) = \lim_{t \rightarrow \infty} \xi_1(t) = \lim_{p \rightarrow 0} p W_{11}(p) (U_0/\delta) / p = \lim_{\alpha \rightarrow 0} \dots, \quad (31)$$

$$d_{31} (h/\delta) U_0 (M_2 + m/2) / (M_1 + M_2 + m) \quad (31)$$

$$\xi_2(\infty) = \lim_{t \rightarrow \infty} \xi_2(t) = \lim_{p \rightarrow 0} p W_{21}(p) (U_0/\delta) / p = \lim_{\alpha \rightarrow 0} \dots, \quad (32)$$

$$d_{31} (h/\delta) U_0 (M_1 + m/2) / (M_1 + M_2 + m) \quad (32)$$

$$\xi_1(\infty) + \xi_2(\infty) = \lim_{t \rightarrow \infty} (\xi_1(t) + \xi_2(t)) = d_{31} (h/\delta) U_0. \quad (33)$$

The static displacement of the faces of the piezoactuator for the transverse piezoelectric effect at  $m \ll M_1$  and  $m \ll M_2$

$$\xi_1(\infty) = \lim_{t \rightarrow \infty} \xi_1(t) = \lim_{p \rightarrow 0} p W_{11}(p) (U_0/\delta) / p = \lim_{\alpha \rightarrow 0} \dots, \quad (34)$$

$$d_{31} (h/\delta) U_0 M_2 / (M_1 + M_2) \quad (34)$$

$$\xi_2(\infty) = \lim_{t \rightarrow \infty} \xi_2(t) = \lim_{p \rightarrow 0} p W_{21}(p) (U_0/\delta) / p = \lim_{\alpha \rightarrow 0} \dots, \quad (35)$$

$$d_{31} (h/\delta) U_0 M_1 / (M_1 + M_2) \quad (35)$$

Let us consider a numerical example of the calculation of static characteristics of the piezoactuator from piezoceramics PZT under the transverse piezoelectric effect at  $m \ll M_1$  and  $m \ll M_2$ . For  $d_{31} = 2.5 \cdot 10^{-10}$  m/V,  $h = 4 \cdot 10^{-2}$  m,  $\delta = 2 \cdot 10^{-3}$  m,  $U = 50$  V,  $M_1 = 10$  kg and  $M_2 = 40$  kg we obtain the static displacement of the faces of the piezoelectric actuator  $\xi_1(\infty) = 200$  nm,  $\xi_2(\infty) = 50$  nm,  $\xi_1(\infty) + \xi_2(\infty) = 250$  nm.

For the description of the piezoactuator for the longitudinal piezoelectric effect for one rigidly fixed face of the transducer at  $M_1 \rightarrow \infty$  we obtain from equation (27) the transfer functions  $W_{21}(p)$  and  $W_{23}(p)$  of the piezoactuator for the longitudinal piezoelectric effect in the following form:

$$W_{21}(p) = \Xi_2(p) / E_3(p) = d_{33} \delta / [M_2 \delta \chi_{33}^E p^2 + \delta \gamma \text{th}(\delta \gamma)], \quad (36)$$

$$W_{23}(p) = \Xi_2(p) / F_2(p) = -\delta \chi_{33}^E / [M_2 \delta \chi_{33}^E p^2 + \delta \gamma \text{th}(\delta \gamma)]. \quad (37)$$

Accordingly, the static displacement  $\xi_2(\infty)$  of the piezoactuator under the longitudinal piezoeffect in the form

$$\xi_2(\infty) = \lim_{t \rightarrow \infty} \xi_2(t) = \lim_{p \rightarrow 0} p W_{21}(p) U_0 / p = d_{33} U_0, \quad (38)$$

$$\xi_2(\infty) = \lim_{p \rightarrow 0} p W_{23}(p) F_0 / p = -\delta s_{33}^E F_0 / S_0. \quad (39)$$

Let us consider a numerical example of the calculation of static characteristics of the piezoactuator under the longitudinal piezoeffects. For  $d_{33} = 5 \cdot 10^{-10}$  m/V,  $U = 500$  V we obtain  $\xi_2(\infty) = 250$  nm. For  $\delta = 6 \cdot 10^{-4}$  m,  $s_{33}^E = 3.5 \cdot 10^{-11}$  m<sup>2</sup>/N,  $F_0 = 2000$  N,  $S_0 = 1.75 \cdot 10^{-4}$  m<sup>2</sup> we obtain  $\xi_2(\infty) = -240$  nm. The experimental and calculated values for the piezoactuator are in agreement to an accuracy of 5%.

Let us consider the operation at low frequencies for the piezoactuator with one face rigidly fixed so that  $M_1 \rightarrow \infty$  and  $m \ll M_2$ . Using the approximation of the hyperbolic cotangent by two terms of the power series in transfer functions (36) and (37), at  $m \ll M_2$  we obtain the expressions in the frequency range of  $0 < \omega < 0.01 c^E/\delta$

$$W_{21}(p) = \Xi_2(p) / E_3(p) = d_{33} \delta / (T_i^2 p^2 + 2 T_i \xi_i p + 1), \quad (40)$$

$$W_{23}(p) = \Xi_2(p)/F_2(p) = -\left(s_{33}^E \delta / S_0\right) / \left(T_i^2 p^2 + 2T_i \xi_i p + 1\right), \quad (41)$$

$$T_i = (\delta / c^E) \sqrt{M_2 / m} = \sqrt{M_2 / C_{33}^E}, \quad \xi_i = (\alpha \delta / 3) \sqrt{m / M_2}, \\ C_{33}^E = S_0 / (s_{33}^E \delta) = 1 / (\chi_{33}^E \delta).$$

where  $T_i$  is the time constant and  $\xi_i$  is the damping coefficient,  $C_{33}^E$  - is the rigidity of the piezoactuator under the longitudinal piezoeffect.

In the static mode of operation the piezoactuator for mechatronic systems and elastic load we obtain the equation the following form

$$\frac{\xi_2}{\xi_{2m}} = \frac{1}{1 + C_e / C_{33}^E}, \quad (42)$$

where  $\xi_2$  is the displacement of the piezoactuator in the case of the elastic load,  $\xi_{2m} = d_{33} U_0$  is the maximum displacement of the piezoactuator,  $C_e$  is the load rigidity.

Equations (40, 42) yield the transfer function of the piezoelectric actuator with a fixed end and elastic inertial load in the following form

$$W_2(p) = \frac{\Xi_2(p)}{U(p)} = \frac{d_{33}}{(1 + C_e / C_{33}^E)(T_i^2 p^2 + 2T_i \xi_i p + 1)}, \quad (43)$$

where the time constant  $T_i$  and the damping coefficient  $\xi_i$  are determined by the formulas

$$T_i = \sqrt{M_2 / (C_e + C_{33}^E)}, \quad \xi_i = \alpha \delta^2 C_{33}^E / \left(3c^E \sqrt{M(C_e + C_{33}^E)}\right).$$

Let us consider the operation at low frequencies for the piezoactuator with one face rigidly fixed so that  $M_1 \rightarrow \infty$  and  $m \ll M_2$  for  $M_2 = 10$  kg,  $C_{33} = 9 \cdot 10^6$  N/m,  $C_e = 10^6$  N/m we obtain  $T_i = 10^{-3}$  s.

## 4. Results and Discussions

We obtain a generalized parametric structural schematic diagram of electromagnetoelastic actuator for mechatronic system on Figure 7 taking into account equation of generalized electromagnetoelasticity (piezoelectric, piezomagnetic, electrostriction, and magnetostriction effects) and decision wave equation. The results of constructing a generalized structural-parametric model and a generalized parametric structural schematic diagram of electromagnetoelastic actuator are shown in Figure 7. The parametric structural schematic diagrams of piezoelectric actuators for longitudinal, transverse, shift piezoelectric effects on Figure 2, Figure 4, Figure 6 are converted to the generalized parametric structural schematic diagram of the electromagnetoelastic actuator Figure 7 with the replacement of the following parameters

$$\Psi_m = E_3, E_3, E_1, \quad v_{mi} = d_{33}, d_{31}, d_{15},$$

$$s_{ij}^\Psi = s_{33}^E, s_{11}^E, s_{55}^E, \quad l = \delta, h, b.$$

From generalized structural-parametric model and generalized parametric structural schematic diagram of the electromagnetoelastic actuator after algebraic transformations we obtain the transfer functions of the electromagnetoelastic actuator. The piezoelectric actuator for the transverse piezoelectric effect compared to the piezoelectric actuator for the longitudinal piezoelectric effect provides a greater range of static displacement and a less range of working force. The magnetostriction actuator has a greater range of the static working force.

It is possible to construct the generalized structural-parametric model, generalized parametric structural schematic diagram and the transfer functions in matrix form of the electromagnetoelastic actuator using the solutions of the wave equation of the actuator and taking into account the features of its deformations along the coordinate axes.

## 5. Conclusions

Using the obtained solution of the wave equation and taking into account the features of the deformations along the coordinate axes, it is possible to construct the generalized structural-parametric model and parametric structural schematic diagram of the electromagnetoelastic actuator for mechatronic system and to describe dynamic and static properties of the actuator.

The transfer functions and the parametric structural schematic diagram of the piezoactuators for longitudinal, transverse, shift piezoeffects are obtained from structural parametric models of the piezoactuators.

The transfer functions in matrix form of the electromagnetoelastic actuator are describe its deformations during operation as a part of the mechatronic system.

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